

For a physical pendulum, we still have

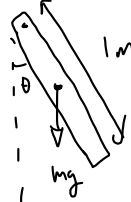
$$\omega = \sqrt{\frac{mgL}{I}}$$

But now $I \neq mL^2$ (what we used for simple)

For the meter stick,

$$L = 0.5 \text{ m,}$$

$$I = \frac{mL^2}{3} \text{ where } L = \text{total length}$$



from table 10.20. In this case $L = \frac{l}{2}$

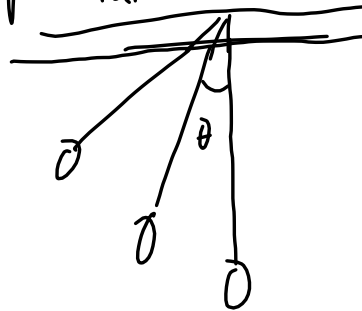
$$\text{So } \omega = \sqrt{\frac{mgL}{\frac{1}{3}m(2L)^2}} = \sqrt{\frac{3g}{4L}} = 3.85 \text{ s}^{-1} \text{ or}$$

$$T = \frac{2\pi}{\omega} = 1.64 \text{ s}$$

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Angular acceleration for a pendulum

$$\alpha = \frac{d^2\theta}{dt^2}$$



For a mass/spring, we have

$$x = A \cos \omega t \text{ where } \omega = \sqrt{\frac{k}{m}}.$$

For pendulum

$$\theta = \theta_{\max} \cos(\omega t) \quad \frac{d^2\theta}{dt^2} = -\theta_{\max} \omega^2 \cos(\omega t)$$

where $\omega = \sqrt{\frac{g}{L}}$ for simple pendulum. Re-note!

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Two springs in series

Total displacement
 $dx_{total} = dx_1 + dx_2$

Hooke's Law
 $F_1 = -k_{eff}(dx_1 + dx_2)$

$F_1 = k_1 dx_1 = k_2 dx_2$
 $dx_2 = \frac{k_1}{k_2} dx_1$

$F_1 = -k_{eff}(dx_1 + \frac{k_1}{k_2} dx_1)$
 $F_1 = -k_1 dx_1$

$F_1 = k_{eff}(1 + \frac{k_1}{k_2}) dx_1 = -k_1 dx_1$

$k_{eff} = \frac{k_1}{1 + \frac{k_1}{k_2}} = \frac{k_1 k_2}{k_1 + k_2}$

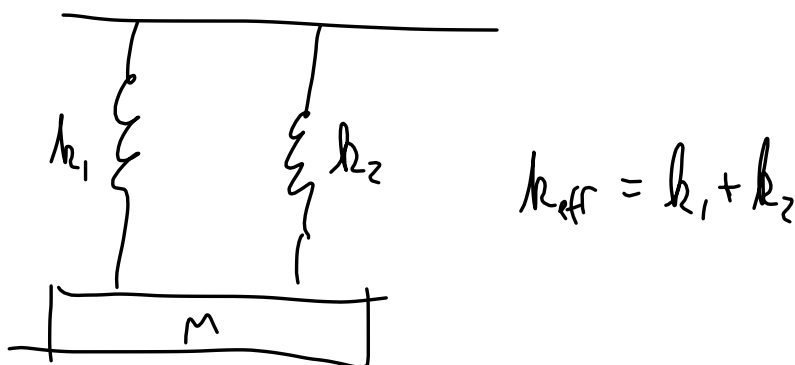
IF $k_1 = k_2$ (ETA problem)

Then $k_{eff} = \frac{k}{2}$

This is similar to
 2 capacitors in series (phys 121)

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Springs in parallel



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Torsional oscillator

$$\theta = \theta_{\max} \cos(\omega t)$$

where $\omega = \sqrt{\frac{K}{I}}$, $K = \text{torsional constant}$
 $I = \text{moment of inertia}$

Cavendish:

$K = \omega^2 I$

We can find K by measuring ω , and

$$I = I_{\text{rod}} + I_{\text{masses}}$$

$$= \frac{1}{12} M_{\text{rod}} L^2 + 2 \left(m_{\text{ball}} \left(\frac{L}{2} \right)^2 \right)$$

measure $M_{\text{rod}}, M_{\text{ball}}, L$

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Damped oscillations

If we consider friction, (hard problem)

$$kx - \mu_k mg = ma$$

$$-kx - \mu_k mg = ma$$

Frictional force changes sign, depending on position

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But if damping depends on velocity, then we have a chance

$$F_{\text{damping}} = -bV_x \quad (\text{viscous fluid, shock absorber})$$

\nwarrow damping constant

$$\Sigma F = -kx - bV_x = -kx - b \frac{dx}{dt} = m \frac{d^2x}{dt^2}$$

$$\frac{d^2x}{dt^2} = -\frac{k}{m}x - \frac{b}{m} \frac{dx}{dt}$$

Solution: $x(t) = A e^{-\left(\frac{b}{2m}\right)t} \cos(\omega t)$

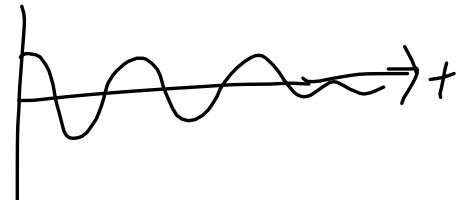
where $\omega = \sqrt{\frac{k}{m} - \left(\frac{b}{2m}\right)^2}$

For ETA problem,
 $E = \frac{1}{2} k A^2$

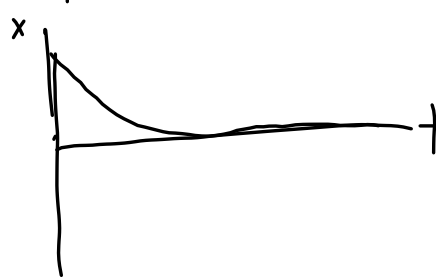
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3 cases for damped oscillators: x

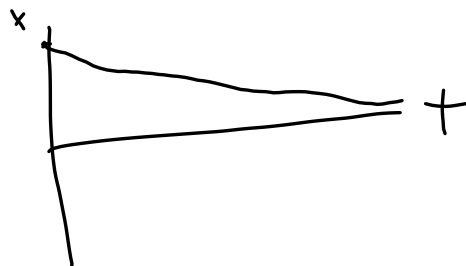
1. $b < 2\sqrt{km}$, underdamping



2. $b = 2\sqrt{km}$ critical damping



3. $b > 2\sqrt{km}$ overdamped



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